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Published in:
Physical Review. B: Condensed Matter and Materials Physics

DOI:
[10.1103/PhysRevB.80.235401](https://doi.org/10.1103/PhysRevB.80.235401)

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Document Version
Publisher's PDF, also known as Version of record

Publication date:
2009

[Link to publication in University of Groningen/UMCG research database](#)

Citation for published version (APA):
van Zwol, P. J., Svetovoy, V. B., & Palasantzas, G. (2009). Distance upon contact: Determination from roughness profile. *Physical Review. B: Condensed Matter and Materials Physics*, 80(23), 235401-1-235401-5. [235401]. <https://doi.org/10.1103/PhysRevB.80.235401>

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(Received 7 October 2009; revised manuscript received 2 November 2009; published 1 December 2009)

The point at which two random rough surfaces make contact takes place at the contact of the highest asperities. The distance upon contact d_0 in the limit of zero load has crucial importance for determination of dispersive forces. Using gold films as an example we demonstrate that for two parallel plates d_0 is a function of the nominal size of the contact area L and give a simple expression for $d_0(L)$ via the surface roughness characteristics. In the case of a sphere of fixed radius R and a plate the scale dependence manifests itself as an additional uncertainty $\delta d(L)$ in the separation, where the scale L is related with the separation d via the effective area of interaction $L^2 \sim \pi R d$. This uncertainty depends on the roughness of interacting bodies and disappears in the limit $L \rightarrow \infty$.

DOI: [10.1103/PhysRevB.80.235401](https://doi.org/10.1103/PhysRevB.80.235401)

PACS number(s): 68.35.Ct, 12.20.Fv, 68.35.Np, 68.37.Ps

I. INTRODUCTION

The absolute distance separating two bodies is a parameter of principal importance for the determination of dispersive forces (van der Waals,¹ Casimir,² or more general Casimir-Lifshitz force³). The absolute distance becomes difficult to determine when the separation gap approaches nanometer dimensions. This complication originates from the presence of surface roughness, which manifests itself on the same scale. In fact, when the bodies are brought into gentle contact they are still separated by some distance d_0 , which we call the distance upon contact due to surface roughness.

We are interested in the dispersive forces when stronger chemical or capillary forces are eliminated. In this case d_0 has a special significance for adhesion, which is mainly due to van der Waals forces across an extensive noncontact area.⁴ The distance d_0 is important for micro (nano) electro mechanical systems (MEMS) because stiction due to adhesion is the major failure mode in MEMS.⁵ Furthermore, the distance upon contact plays an important role in contact mechanics⁶ is very significant for heat transfer,⁷ contact resistivity,⁸ lubrication, and sealing.⁹ In addition, it has also importance in the case of capillary forces and wetting,^{10–12} where knowledge of d_0 provides further insight of how adsorbed water wets a rough surface.

The distance upon contact d_0 between a sphere and a plate^{13,14} plays a key role in modern precise measurements of the dispersion forces (see Ref. 15 for a review) where d_0 is the main source of errors. In Casimir force measurements d_0 is determined using electrostatic calibration. In this case the force dependence on the separation is known, and one can determine the absolute separation (see recent discussions^{16–18}). Even when the distance is not counted from the point of contact^{16,17,19} local realization of roughness as shown in this paper will contribute to uncertainty of the absolute separation.

Independent attempts to define d_0 were undertaken in experiments measuring the adhesion energy.⁴ It was proposed²⁰ to take d_0 as the sum of the root-mean-square (rms) roughnesses of two surfaces upon contact. This definition is, however, restricted and can only be used for rough estimates as stressed in Ref. 20. Obviously, the distance upon contact has to be defined by the highest asperities.

In this paper we present a simple method for determination of d_0 from the roughness profiles of the two surfaces coming into contact. For two plates it is explicitly demonstrated that $d_0(L)$ is scale dependent, where L^2 is the area of nominal contact. We discuss also application of our method to the sphere-plate configuration. In this case it is shown that d_0 determined from the electrostatic calibration can differ from that playing role in the dispersive force and the difference is scale (separation) dependent.

In Sec. II we report briefly the details of our film preparation and characterization. In Sec. III the roughness profiles in the plate-plate configuration are discussed and the main relation connecting d_0 with the size of the nominal contact is deduced. The sphere-plate configuration is discussed in Sec. IV together with uncertainty in d_0 . Our conclusions are collected in Sec. V.

II. EXPERIMENTAL

The surfaces we use in this study were gold films grown by thermal evaporation onto oxidized silicon wafers with thicknesses in the range 100–1600 nm and having different rms roughness. A polystyrene sphere (radius $R=50 \mu\text{m}$), attached on a gold coated cantilever, was first plasma sputtered with gold for electrical contact, and then a 100 nm gold film grown on top of the initial coating. The deposited films were of uniform thickness and of isotropic surface morphology as was confirmed independently with atomic force and scanning electron microscopy on different locations.

The surface profile was recorded with Veeco Multimode atomic force microscope (AFM) using Nanoscope V controller. To analyze the effect of scale dependence, megascans of large area up to $40 \times 40 \mu\text{m}^2$ were made and recorded with the lateral resolution of 4096×4096 pixels. The maximal area, which we have been able to scan on the sphere, was $8 \times 8 \mu\text{m}^2$ (2048×2048 pixels). All images were flattened with linear filtering; for the sphere the parabolic filtering was used to exclude the effect of curvature. Figure 1 shows the images of the 100 nm film (a) and the sphere (b) on different scales. Approximately 10 images of smaller size $500 \times 500 \text{ nm}^2$ were recorded for each film and for the sphere to

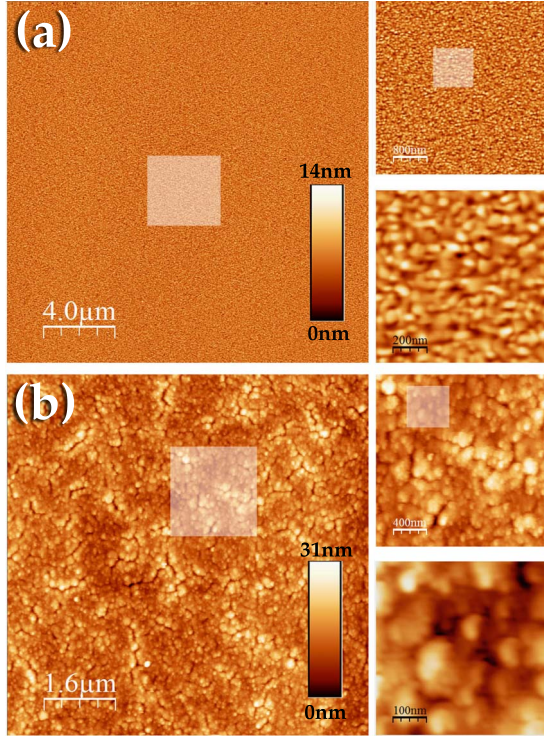


FIG. 1. (Color online) AFM megascan of the 100 nm film (a) and the sphere (b). The insets show the highlighted areas at higher magnifications.

obtain the correlation length ξ of the rough surfaces.²¹ Finally, the electrostatic calibration was used for the determination of the cantilever spring constant and d_0 .²²

III. PLATE-PLATE CONTACT

Consider first two parallel plates, which can come into contact. A plate surface can be described by a roughness profile $h_i(x, y)$ ($i=1, 2$ for body 1 or 2), where x and y are the lateral coordinates. The averaged value over large area of the profile is zero, $\langle h_i(x, y) \rangle = 0$. Then the local distance between the plates is

$$d(x, y) = d - h_1(x, y) - h_2(x, y), \quad (1)$$

where d is the distance between the average planes. We can define the distance upon contact d_0 as the largest distance $d = d_0$, for which $d(x, y)$ becomes zero.

It is well known from contact mechanics²³ that the contact of two elastic rough plates is equivalent to the contact of a rough hard plate and an elastic flat plate with an effective Young's modulus E and a Poisson ratio ν . In this paper we analyze the contact in the limit of zero load when both bodies can be considered as hard. This limit is realized when only weak adhesion is possible, for which the dispersive forces are responsible. Strong adhesion due to chemical bonding or due to capillary forces is not considered here. This is not a principal restriction, but the case of strong adhesion has to be analyzed separately. Equation (1) shows that the profile of the effective rough body is given by

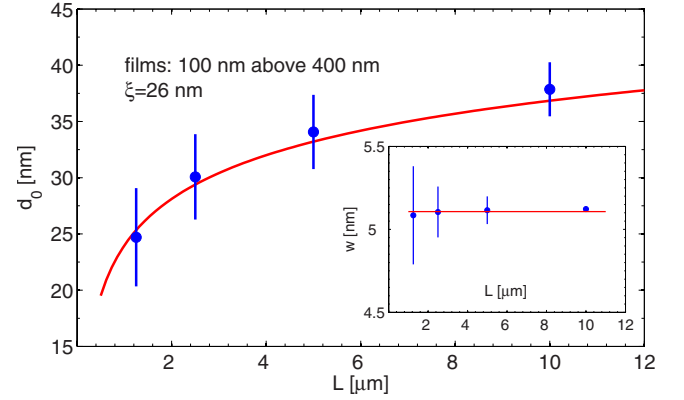


FIG. 2. (Color online) Distance upon contact as a function of the length scale. Dots with the error bars are the values calculated from the megascans. The solid curve is the theoretical expectation according to Eq. (4). The inset demonstrates absence of the scale dependence for the rms roughness.

$$h(x, y) = h_1(x, y) + h_2(x, y). \quad (2)$$

The latter means that $h(x, y)$ is given by the combined image of the surfaces facing each other.

Let L_0 be the size of the combined image. Then, in order to obtain information on the scale $L = L_0/2^n$, we divide this image on 2^n subimages. For each subimage we find the highest point of the profile (local d_0), and average all these values. This procedure gives us $d_0(L)$ and the corresponding statistical error. Megascans are very convenient for this purpose otherwise one has to collect many scans in different locations.

For the 100 nm film above the 400 nm film the result of this procedure is shown in Fig. 2. We took the maximum area to be $10 \times 10 \mu\text{m}^2$. The figure clearly demonstrates the dependence of d_0 on the scale L although the errors appear to be significant. The inset shows the dependence of the rms roughness w on the length scale L . This dependence is absent in accordance with the expectations, while only the error bars increase when L is decreasing.

To understand the dependence $d_0(L)$ let us assume that the size L of the area of nominal contact is large in comparison with the correlation length, $L \gg \xi$. It means that this area can be divided into a large number $N^2 = L^2/\xi^2$ of cells. The height of each cell (asperity) can be considered as a random variable h .²⁴ The probability to find h smaller than some value z can be presented in a general form

$$P(z) = 1 - e^{-\phi(z)}, \quad (3)$$

where the "phase" $\phi(z)$ is a nonnegative and nondecreasing function of z . Note that Eq. (3) is just a convenient way to represent the data: instead of cumulative distributions $P(z)$ we are using the phase $\phi(z)$.

For a given asperity the probability to find its height above d_0 is $1 - P(d_0)$, then within the area of nominal contact one asperity will be higher than d_0 if

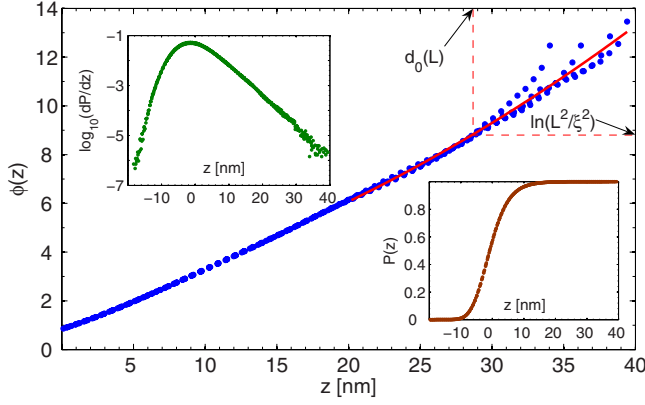


FIG. 3. (Color online) Statistics of the surface roughness. Four $10 \times 10 \mu\text{m}^2$ images were used. The main graph shows the "phase" as a function of z . The continuous red (light gray) curve is the best fit of the data at large z and the dashed lines demonstrate the solution of Eq. (4). The top inset presents the logarithm of the density function. The bottom inset shows the cumulative distribution.

$$e^{-\phi(d_0)(L^2/\xi^2)} = 1 \quad \text{or} \quad \phi(d_0) = \ln(L^2/\xi^2). \quad (4)$$

This condition can be considered as an equation for the asperity height because due to a sharp exponential behavior the height is approximately equal to d_0 . To solve Eq. (4) we have to know the function $\phi(z)$, which can be found from the roughness profile.

The cumulative distribution $P(z)$ can be found from a roughness profile by counting pixels with the height below z . Then the "phase" can be calculated as $\phi(z) = -\ln(1-P)$. The results are presented in Fig. 3. It has to be noted that the function $\phi(z)$ becomes more dispersive at large z . This effect was observed for all surfaces we investigated. To solve Eq. (4) we have to approximate the large z tail of $\phi(z)$ by a smooth curve. Any way of the data smoothing is equally good, and our method is not relied on specific assumptions about the probability distribution. The procedure of solving Eq. (4) is shown schematically in Fig. 3, and the solution itself is the continuous red (light gray) curve in Fig. 2.

It has to be mentioned that the normal distribution fails to describe the data at large z . Other known distributions are not able satisfactory describe the data at all z . Asymptotically at large z the data can be reasonably well fit with the generalized extreme value distributions Gumbel or Weibull.^{25,26} This

fact becomes important if one has to know d_0 for the size L , which is larger than the maximal scan size. In this case one has to extrapolate $\phi(z)$ to large z according to the chosen distribution. In this paper we are not doing extrapolation using only $\phi(z)$ extracted directly from the megascans.

The observed dependence $d_0(L)$ can be understood intuitively. The probability to have one high asperity is exponentially small but the number of asperities increases with the area of nominal contact. Therefore, the larger the contact area, the higher probability to find a high feature within this area.

Our result found in the limit of zero load will hold true if the elastic deformation of the highest asperity will be small ($\ll d_0$). Applying Hertzian theory to an asperity of radius $\xi/2$ one finds the restriction on the load p ,

$$p \ll \sqrt{2\xi/9}d_0(1-\nu^2)^{-1}(d_0^2/L^2)E. \quad (5)$$

If $p = A_H/6\pi d_0^3$ is the van der Waals pressure (A_H is the Hamaker constant) then (5) for the *Au* parameters restricts d_0 and L as $(d_0/10\text{nm})^{4.5}(L/10\mu\text{m})^{-2} \gg 0.3$. This condition is true in the range of main interest. For the sphere-plane case (see below) Eq. (5) can be modified accordingly but in general the physical contact is not assumed for the sphere-plate configuration.

IV. SPHERE-PLATE CONTACT

The other question of great practical importance is the distance upon contact between a sphere and a plate. In the experiments^{13,14,17,19,22} the sphere attached to a cantilever or an optical fibre approaches the plate. Assuming that the sphere is large, $R \gg d$, the local distance is

$$d(x,y) = d + (x^2 + y^2)/2R - h(x,y), \quad (6)$$

where $h(x,y)$ is the combined profile of the sphere and the plate.

Again, d_0 is the maximal d , for which the local distance becomes zero. This definition gives

$$d_0 = \max_{x,y} [h(x,y) - (x^2 + y^2)/2R]. \quad (7)$$

In contrast with the plate-plate configuration now d_0 is a function of the sphere radius R , but, of course, one can define the length scale L_R corresponding to this radius R (see below).

TABLE I. The parameters characterizing the sphere-film systems (all in nm). The first five rows were determined from combined images (see text). The last row d_0^{el} gives the values of d_0 determined electrostatically. The last four rows were determined for $R=50 \mu\text{m}$.

	100 nm	200 nm	400 nm	800 nm	1600 nm
w	3.8	4.2	6.0	7.5	10.1
ξ	26.1 ± 3.8	28.8 ± 3.7	34.4 ± 4.7	30.6 ± 2.4	42.0 ± 5.5
L_R	920	1050	1470	1560	2100
d_0^{th}	12.5	14.0	22.8	31.5	53.0
d_0^{im}	12.8 ± 2.2	15.9 ± 2.7	24.5 ± 4.8	31.3 ± 5.4	55.7 ± 9.3
d_0^{el}	17.7 ± 1.1	20.2 ± 1.2	23.0 ± 0.9	34.5 ± 1.7	50.8 ± 1.3

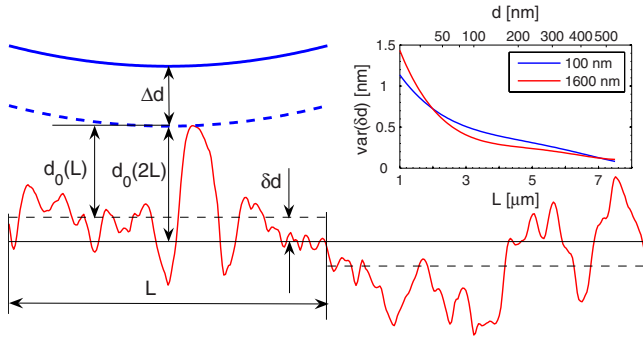


FIG. 4. (Color online) Schematic explanation of additional uncertainty δd in d_0 (see text). The sphere in two positions is shown by the dashed (contact) and solid blue (dark gray) curves. The inset shows the variance of δd as a function of the scale L or separation d .

As input data in Eq. (7) we used the combined images of the sphere and different plates. The origin ($x=0$, $y=0$) was chosen randomly in different positions and then d_0 was calculated according to Eq. (7). We averaged d_0 found in 80 different locations to get the values of d_0^{im} , which are collected in Table I.

We can estimate the same value theoretically. A circle of a finite area L^2 is important in Eq. (7). Asperities of the size ξ are distributed homogeneously within this circle. Then the averaged value of the second term in Eq. (7) is $L^2/4\pi R$. The averaged maximal value of $h(x,y)$ is the distance upon contact between two plates of the size L . This distance is the solution of Eq. (4). In this section we will denote it as $d_0^{pp}(L)$ not to mix with d_0 in the sphere-plate configuration. Then one can find d_0 for the sphere-plate contact by maximizing Eq. (7) on L ,

$$d_0 = \max_L [d_0^{pp}(L) - L^2/4\pi R]. \quad (8)$$

The solution of this equation defines d_0^{th} and the scale L_R corresponding to the maximum. The values of d_0^{th} and L_R found from Eq. (8) are given in Table I for the radius $R = 50 \mu\text{m}$.

One can see that d_0^{th} is in agreement with d_0^{im} determined from the combined images. Comparing it with the values d_0^{el} determined electrostatically one sees that in the first two columns the values of d_0^{el} are considerably larger. Moreover, the errors in d_0^{el} are smaller than in d_0^{im} .

We described d_0 as the value determined from the area L_R^2 and averaged over its different locations. Determination of d_0 from the electrostatic measurements did not undergo this type of averaging. As a result it is sensitive to the local roughness realization near the contact location. This explains why the errors in d_0^{el} are smaller: statistical variation of d_0 from place to place is not included in the errors of d_0^{el} .

Very different local values of d_0 can be found, and for this reason d_0^{el} can deviate significantly from the mean value. Choosing arbitrarily the contact locations in the image of the sphere and the 100 nm film we found, for example, that about 5% of the cases are in agreement with the measured value $d_0^{el} = 17.7 \pm 1.1 \text{ nm}$. One can imagine that the place of contact on the sphere has at least one asperity above the

average. In the combined image the sphere dominates since it is rougher than the film, $w_{sph} = 3.5 \text{ nm}$ and $w_{100} = 1.5 \text{ nm}$. Because the sphere is rigidly fixed on the cantilever the same feature will be in the area of contact for any other location or other film. Already for the sphere above 400 nm film the high feature on the sphere will not play significant role because the roughness of the film, $w_{400} = 4.9 \text{ nm}$, is higher than that for the sphere. In this case we would expect that d_0^{el} has to be in agreement with the averaged value found from the image that is precisely what happens.

Consider now the experimental situation when the dispersive force is measured in the sphere-plate configuration. The system under consideration is equivalent to a smooth sphere above a combined rough profile $h(x,y)$. The position of the average plane depends on the area of averaging L^2 especially for small scales L . The profile shown in Fig. 4 demonstrates different mean values in the left and right segments shown by the dashed black lines. Both of these values deviate from the middle line for the scale $2L$ (solid black line). The true average plane is defined for $L \rightarrow \infty$.

From Fig. 4 one can see that d_0 for L and $2L$ differ on $\delta d = d_0(L) - d_0(2L)$. To be more precise we can define the uncertainty in d_0 as $\delta d(L) = d_0(L) - d_0$, where we understand d_0 as the value counted from the true average plane ($L \rightarrow \infty$). The distance between bodies is then $d = d_0 + \delta d(L) + \Delta d$, where Δd is the displacement from the contact point. The scale L is defined by the effective area of interaction $L^2 = \alpha \pi R d$ ($\alpha = 2$ for the electrostatic and $\alpha = 2/3$ for the pure Casimir force). Suppose that d_0 found from the electrostatic calibration can be considered as a true value (the electrostatic scale is large, $L_{el} \rightarrow \infty$) then in the dispersive force measurement the bodies are separated by $d = d_0 + \delta d(L_{dis}) + \Delta d$ with the related scale $L_{dis} = \sqrt{\alpha \pi R d}$.

For a fixed L the uncertainty δd is a random variable distributed roughly normally around $\delta d = 0$. However, it has to be stressed that δd manifests itself not as a statistical error but rather as a kind of a systematic error. This is because at a given lateral position of the sphere this uncertainty takes a fixed value. The variance of δd is defined by the roughness statistics. It was calculated from the images and shown as inset in Fig. 4. One has to remember that with a probability of 30% the value of δd can be larger than that shown in Fig. 4.

V. CONCLUSIONS

In conclusion, it is shown that the distance upon contact depends on the lateral size of contacting plates and a simple formula describing $d_0(L)$ is presented. For the sphere and plate an additional uncertainty in the absolute separation d is revealed arising due to variation of the average plane position with the effective area of interaction or equivalently with the separation. Its magnitude depends on the roughness of interacting bodies.

We acknowledge helpful discussions with S. Lamoreaux and R. Onofrio. The research was carried out under Project No. MC3.05242 in the framework of the Strategic Research Programme of the Materials Innovation Institute M2i [the former Netherlands Institute for Metals Research (NIMR)]. The authors benefited from exchange of ideas by the ESF Research Network CASIMIR.

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